

Duct Leakage: Measured Magnitudes and Calculated Impacts

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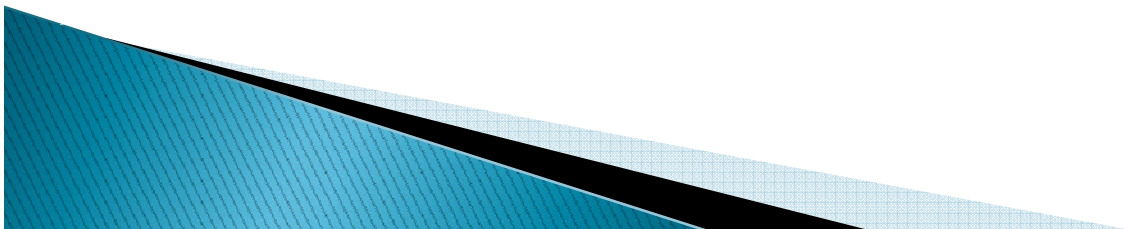
Seminar 16

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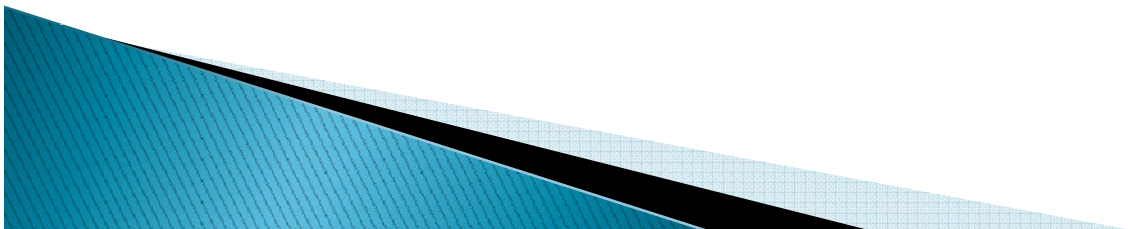
Presentation Overview

- Leakage Area vs. Leakage Flow
 - Measurement techniques
- Are low-pressure system leaks important?
 - Light commercial systems – RTUs
 - Supply sections downstream of VAV boxes
 - Kitchen/bath exhaust systems
- Diagnostic Tools, Leakage Magnitudes, Energy Impacts



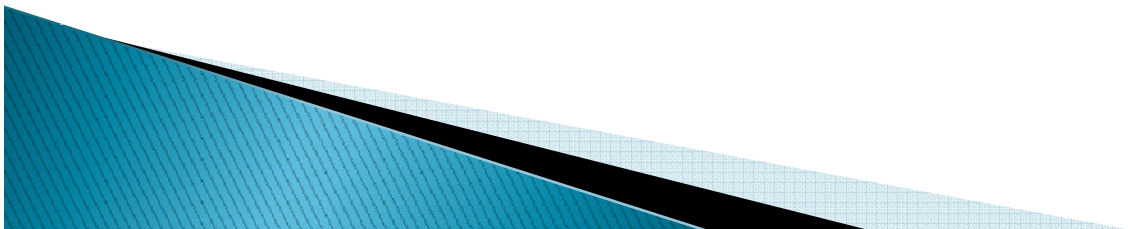
Duct Leakage Characterization

- **Leakage Area vs. Leakage Flow**
 - Leakage Area = size of hole
 - Sometimes expressed as cfm@1"H₂O or cfm@25Pa
 - Needs to be combined with leak pressures for performance analysis
 - Need to separate “low-pressure” and “high-pressure” sections (AT A MINIMUM)
 - Leakage Flow – cfm during normal operation
 - Performance impacts scale with leakage flow
 - % leakage is very useful – remains relatively constant with changes in system flow



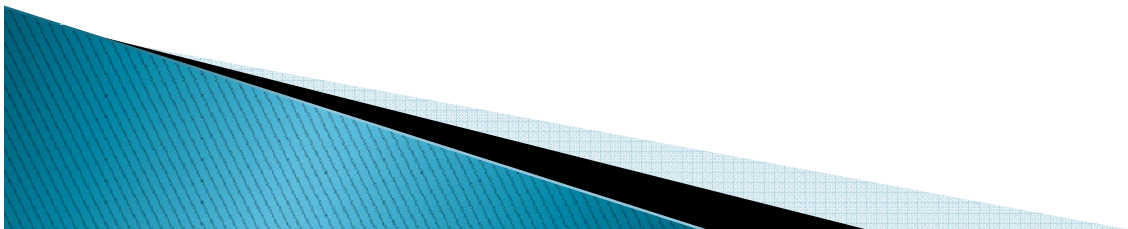
Duct Leakage Measurement

- **“Standard” Fan Pressurization**
 - Measures leakage area
 - Small duct systems (e.g. RTUs)
 - Isolated duct sections in large systems
- **Test and Balance**
 - Measures leakage flow
 - Compares grille flows to fan flow or “design” flow
- **Simplified Leakage Diagnosis or Screening**
 - Kitchen and toilet exhaust
 - Leakage downstream of VAV boxes



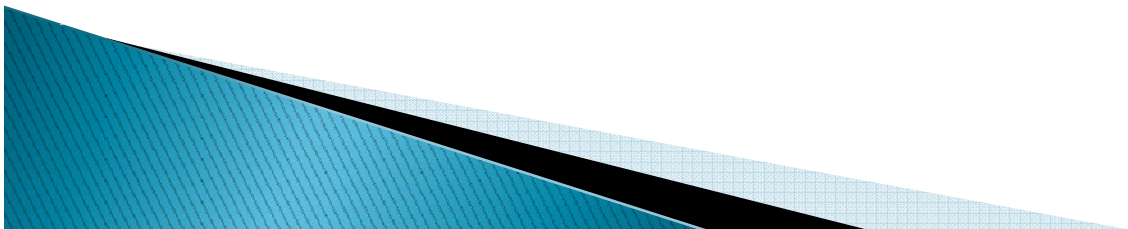
Duct Leakage Magnitude – RTUs

- **Light Commercial Buildings – RTU Systems**
 - Leakage Area data from Fan Pressurization
 - No measured pressures during normal operation
 - 364 systems – all located in Southern California
 - System size: average 3.9 tons (Std Deviation 2.1 tons)
 - Sample: Customers of two Commercial Service Contractors



Duct Leakage Magnitude – RTUs

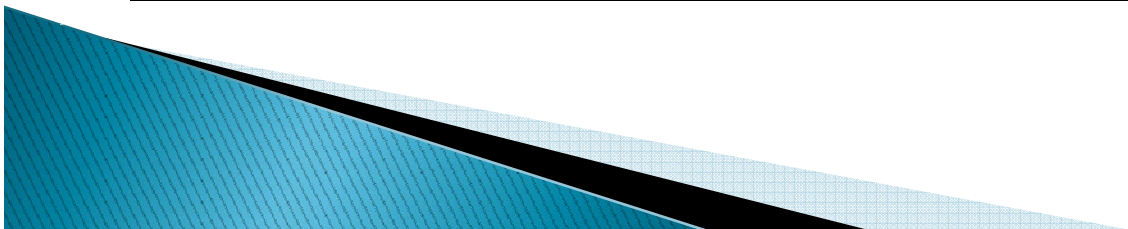
Parameter	Leakage Area [cfm@25/ ton]	Fractional Leakage (Supply and Return) assuming 25 Pa avg at leaks	Fractional Leakage (Supply and Return) assuming 40 Pa avg at leaks
Count [systems]	364	364	364
Average	87	25%	33%
Std Deviation [%]	50%	50%	50%
Std Error in Mean [%]	2.6%	2.6%	2.6%



Energy Impact – RTUs

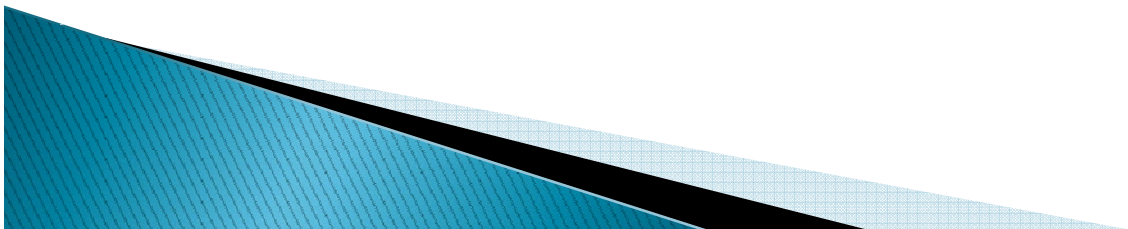
- **Approximate Analysis with ASHRAE Std 152**
 - Thermal losses are primary factor
 - Magnitude depends on location of ducts relative to insulation
 - For ducts above insulation, leakage split evenly between supply and return:

Duct Leakage	25%	33%
Heating Energy Increase	16%	23%
Cooling Energy Increase	28%	40%
Cooling Demand Increase	44%	68%

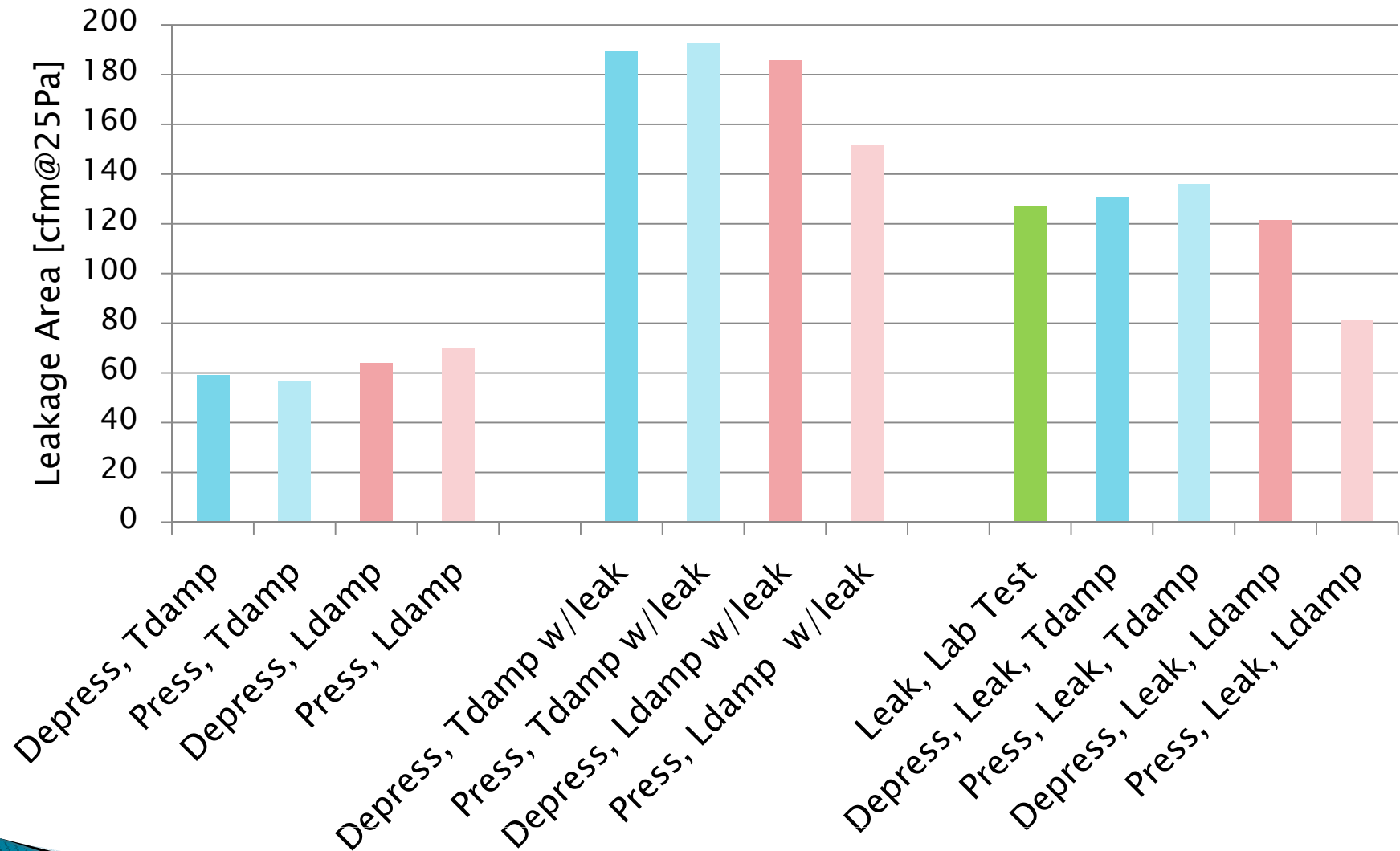


Leakage Diagnosis – VAV Supply

- **Leakage Downstream of VAV Boxes**
 - Standard fan pressurization
 - Test and Balance
 - Generally “low–pressure” ductwork and components
- **Simplified Diagnostic**
 - Leave fan running normally
 - Tape or block all grilles but one
 - VAV damper: close or set to minimum position
 - Pressurize/depressurize through open grille
 - Simultaneously calculate VAV damper opening and downstream leakage

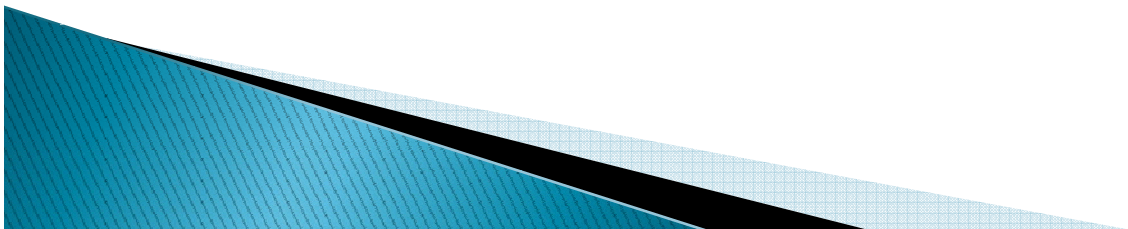


Simplified Diagnostic – VAV Supply



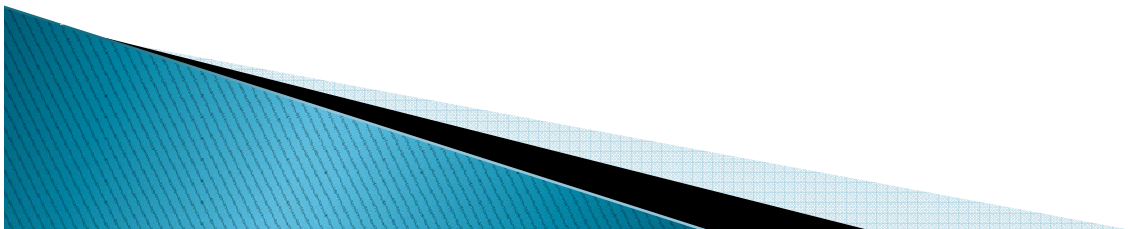
Leakage Magnitude – VAV Supply

- **Supply Leakage Downstream of VAV Boxes**
 - Leakage Area data from Simplified Diagnostic
 - Measured/estimated average downstream pressures during normal operation
 - 9 systems, located California, Florida, Rhode Island, Texas, Washington
 - Vintage: 1980s, 1990s
 - Sample: Office buildings, mostly military and university



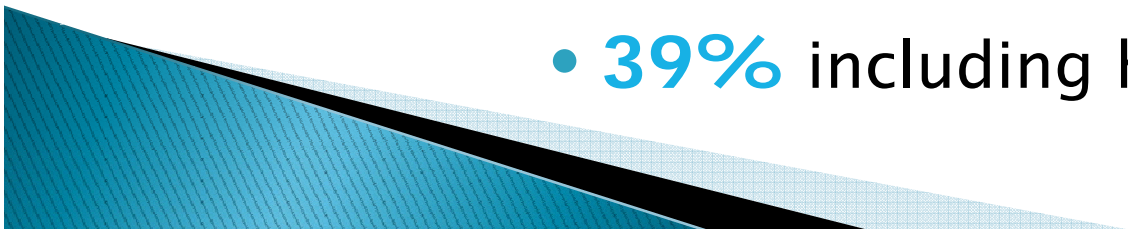
Leakage Magnitude –VAV Supply

Building	Flow Exponent	Best Estimate Leakage	Upstream ΔP [Pa]	Best Estimate ΔP leak [Pa]	Minimum (at grille) ΔP leak [Pa]	State
1	0.80	8%	250	25	8	CA
2	0.72	15%	375	25	10	WA
3	0.54	14%	300	25	20	RI
4	0.64	11%	108	25	15	RI
5	0.61	19%	550	50	50	FL
6	0.61	6%	155	40	10	TX
7	0.78	4%	155	40	10	TX
8	0.53	9%	375	67	9	CA
9	0.41	6%	488	50	20	CA
Average	0.63	10%	306	39	17	
Standard Dev [%]	20%	47%	50%	39%	79%	
Std Err in Mean [%]	7%	16%	17%	13%	26%	



Energy Impact – VAV Supply

- **Fan Power**
 - Varies with flow rate raised to power 2.4 for typical supply systems
 - Duct leaks \Rightarrow short circuit of supply air to the return plenum \Rightarrow excess air flow through fan to meet loads
- **Heating/Cooling Energy**
 - Cooling load due to extra fan heat
 - Heating/cooling loads due to excess outdoor air
 - Ceiling loads during simultaneous heating and cooling
- **Implication of 10% supply leakage:**
 - **29%** excess fan power
 - **39%** including heating/cooling impact



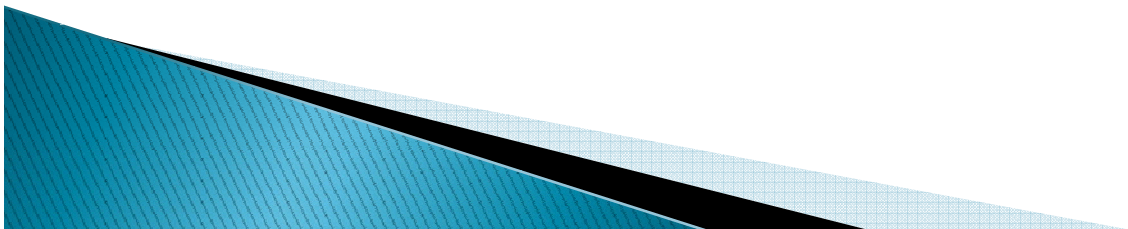
Leakage Diagnosis – Exhaust

- **Kitchen and Toilet Exhaust Systems**

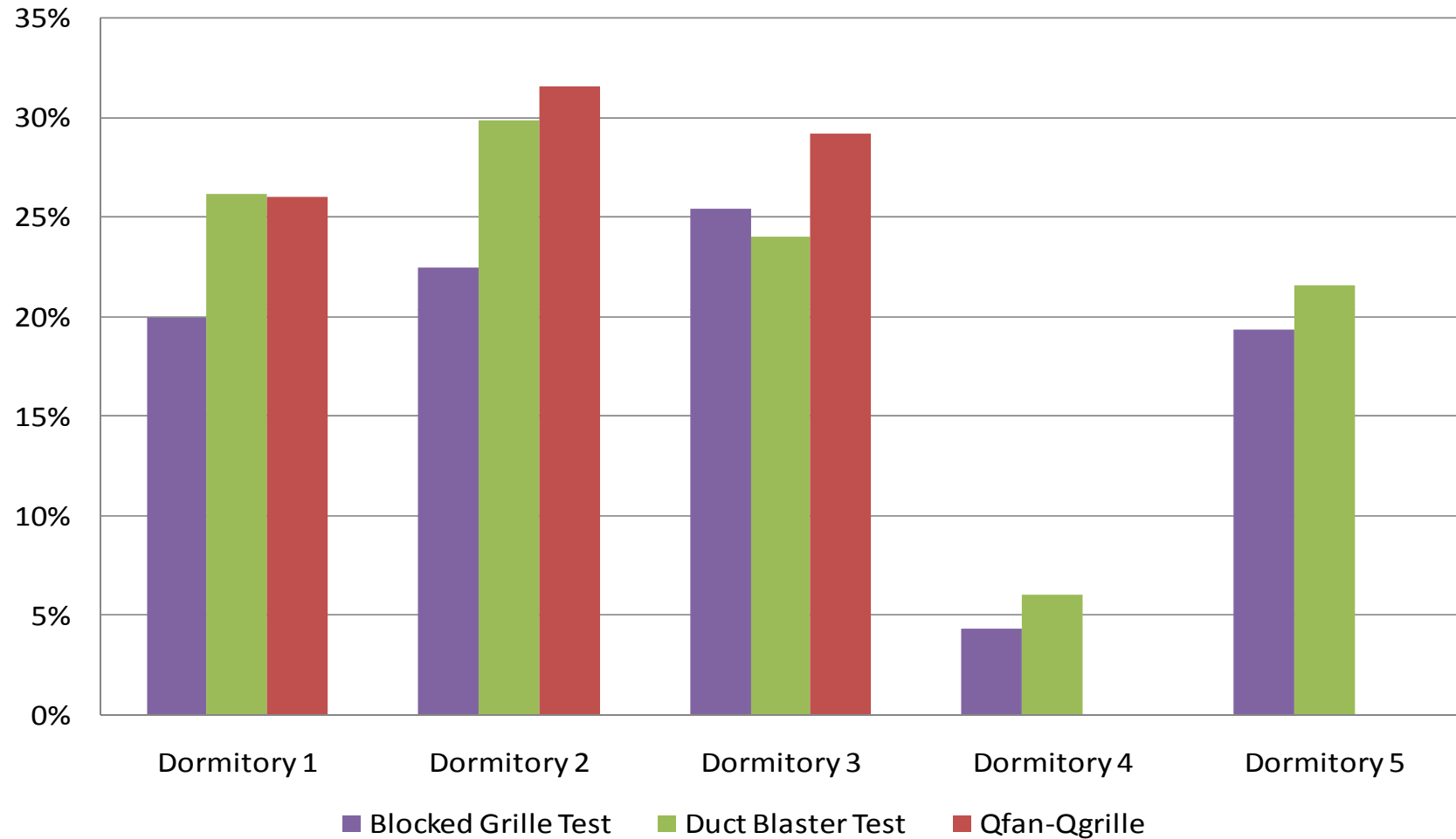
- Standard fan pressurization
- Test and Balance
- Generally “low-pressure”

- **Simplified Diagnostic**

- Leave fan running normally
- Tape or block all grilles
- Measure duct pressure at mid-point of shaft
- Measure flow leaving roof cap

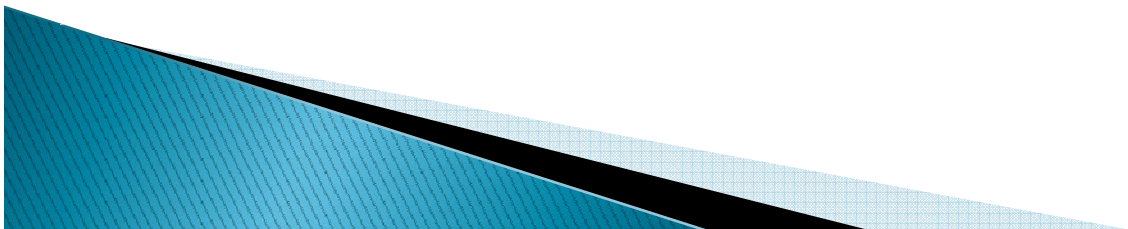


Bath Exhaust Leakage – Simplified Diagnostic



Kitchen/Bath Exhaust Leakage – “Standard” Fan Pressurization

Building	Fan Flow [cfm]	Leakage [%]	Notes
Condominium (40-Story)	950	74%	Building-Cavity Bathroom Exhaust
NYS University Dorm (10-story)	2,300	70%	Bath/Shower Exhaust
NYS University Dorm (7-story)	2,050	54%	Bath/Shower Exhaust
Navy BEQ (10-story dorm)	6,470	54%	Building-Cavity Exhaust w/heat wheel
Barracks (eight 3-story buildings)	20,000	20%	Bath/Shower Exhaust
Office Toilet Exhaust (3-story)	8,700	9%	No pre-diagnosis of leakage
Seven NYC Apartment Exhausts	2,450	36%	Kitchen/Bath Exhausts
Flow-Weighted AVERAGE for 20 Buildings		29%	Based upon leakage area and average pressure differential



Energy Impacts: Kitchen/Bath Exhaust

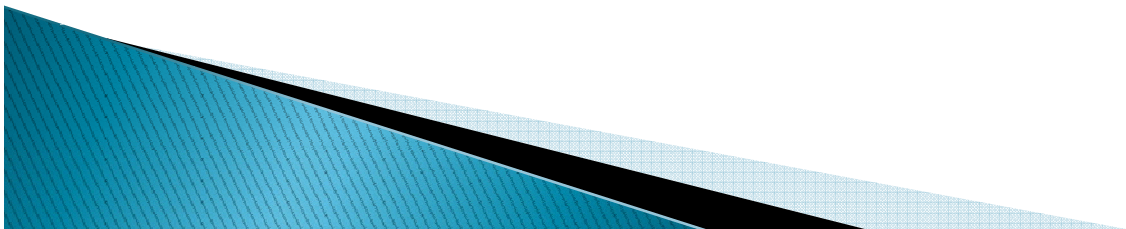
- **Fan Power**

- Scales with Cube of Flow
- **29%** Leakage Increases Fan Power by **179%** to Produce Design Flow at Grilles

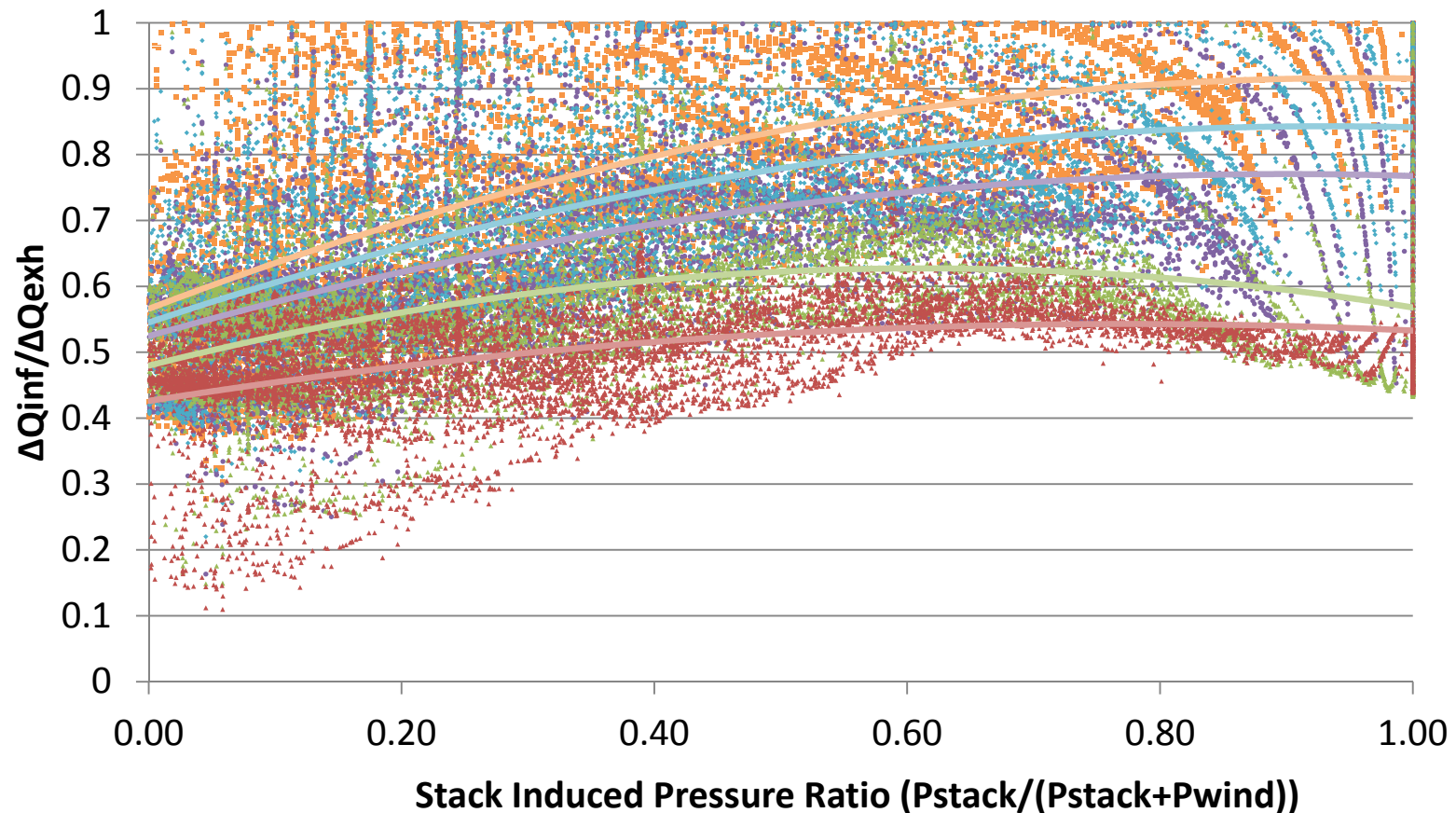
- **Heating and Cooling Loads**

- Scale with Excess Air Infiltration
- Change in infiltration is not equal to reduction in exhaust flow

$$\frac{\Delta Q_{\text{inf}}}{\Delta Q_{\text{exh}}} = \frac{Q_{\text{inf}}^{\text{initial}} - Q_{\text{inf}}^{\text{final}}}{Q_{\text{exh}}^{\text{initial}} - Q_{\text{exh}}^{\text{final}}}$$



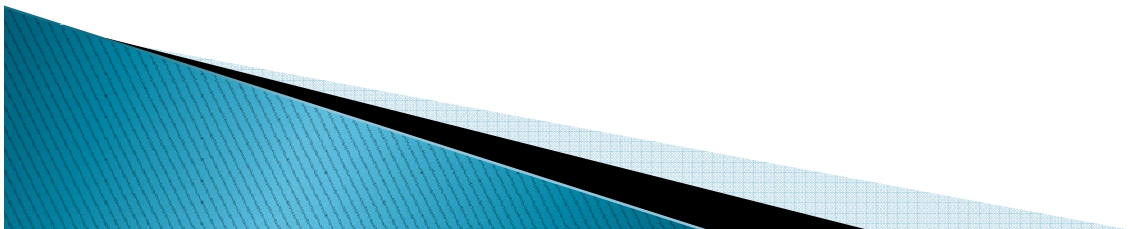
Exhaust Model Results: 6-Story Building



- $dQ_{exh}=1800 \text{ cfm}-1350 \text{ cfm}$
- $dQ_{exh}=1800 \text{ cfm}-900 \text{ cfm}$
- $dQ_{exh}=450 \text{ cfm}-225 \text{ cfm}$
- $n=2$ Polynomial Fit for 1350 cfm - 900 cfm
- Poly. ($dQ_{exh}=900 \text{ cfm}-450 \text{ cfm}$)
- $dQ_{exh}=1350 \text{ cfm} - 900 \text{ cfm}$
- $dQ_{exh}=900 \text{ cfm}-450 \text{ cfm}$
- $n=2$ Polynomial Fit for 1800 cfm - 1350 cfm
- $n=2$ Polynomial Fit for 1800 cfm - 900 cfm
- Poly. ($dQ_{exh}=450 \text{ cfm}-225 \text{ cfm}$)

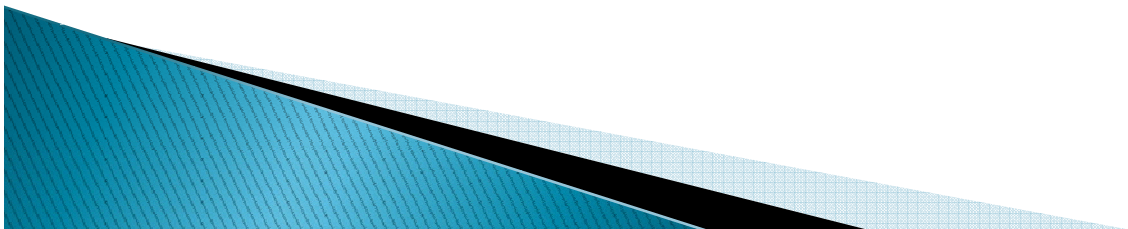
Energy Impact – Kitchen/Bath Exhaust

- **Heating/Cooling Energy**
 - Assumptions
 - 75 cfm nominal kitchen plus bath exhaust for 1000 ft² apartment
 - 0.2 ACH natural infiltration
- **Implication of 29% exhaust leakage:**
 - **179%** excess fan power
 - **23%** increase in ventilation heating/cooling load



Energy Impact Summary

- **Light Commercial RTUs**
 - 15%-70% increase in heating and cooling loads for ducts above ceiling insulation
- **Supply leakage downstream of VAVs**
 - 30%-40% increase in system fan power
- **Kitchen and bath exhaust leakage**
 - >150% increase in fan power
 - >20% increase in infiltration load



Conclusions

- Low-pressure duct leakage is common and substantial
- Simplified techniques exist for measuring/diagnosing low-pressure leakage
- Energy impacts and analysis depend on type of system
 - Light Commercial RTUs
 - Supply leakage downstream of VAVs
 - Kitchen and bath exhaust leakage
- Importance of testing should be based upon leakage percentage of flow, not operating pressure

