



Duct Leakage: Measured Magnitudes and Calculated Impacts

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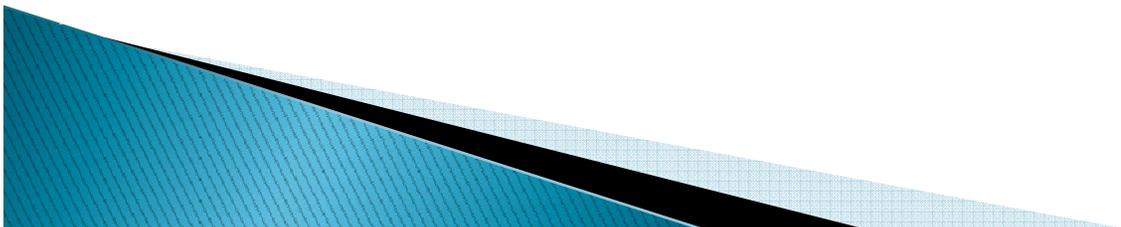
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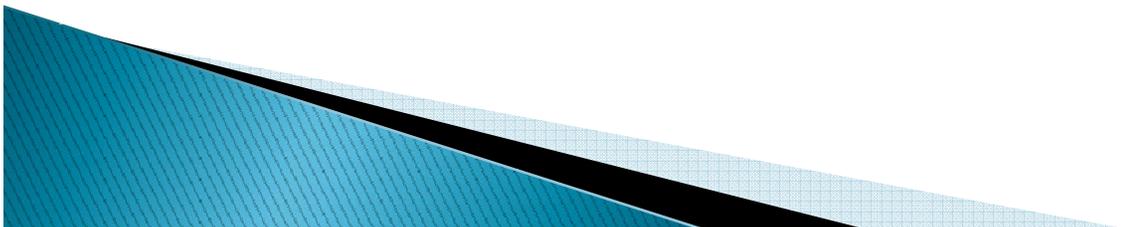
Presentation Overview

- Leakage Area vs. Leakage Flow
 - Measurement techniques
- Are low-pressure system leaks important?
 - Light commercial systems – RTUs
 - Supply sections downstream of VAV boxes
 - Kitchen/bath exhaust systems
- Diagnostic Tools, Leakage Magnitudes, Energy Impacts



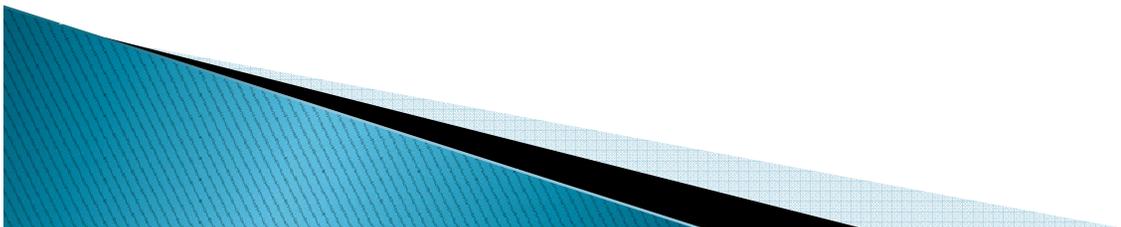
Duct Leakage Characterization

- **Leakage Area vs. Leakage Flow**
 - Leakage Area = size of hole
 - Sometimes expressed as cfm@1”H₂O or cfm@25Pa
 - Needs to be combined with leak pressures for performance analysis
 - Need to separate “low-pressure” and “high-pressure” sections (AT A MINIMUM)
 - Leakage Flow – cfm during normal operation
 - Performance impacts scale with leakage flow
 - % leakage is very useful – remains relatively constant with changes in system flow



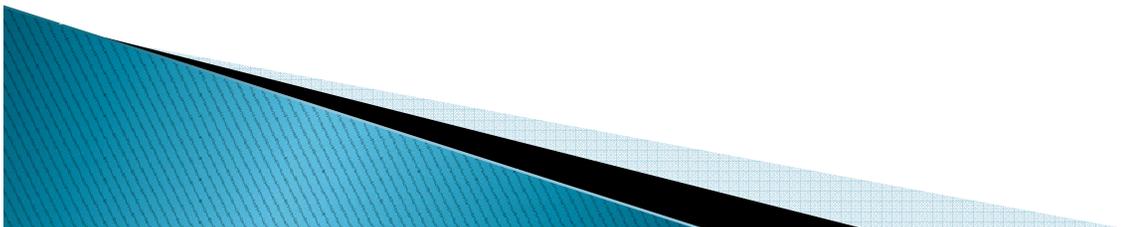
Duct Leakage Measurement

- **“Standard” Fan Pressurization**
 - Measures leakage area
 - Small duct systems (e.g. RTUs)
 - Isolated duct sections in large systems
- **Test and Balance**
 - Measures leakage flow
 - Compares grille flows to fan flow or “design” flow
- **Simplified Leakage Diagnosis or Screening**
 - Kitchen and toilet exhaust
 - Leakage downstream of VAV boxes



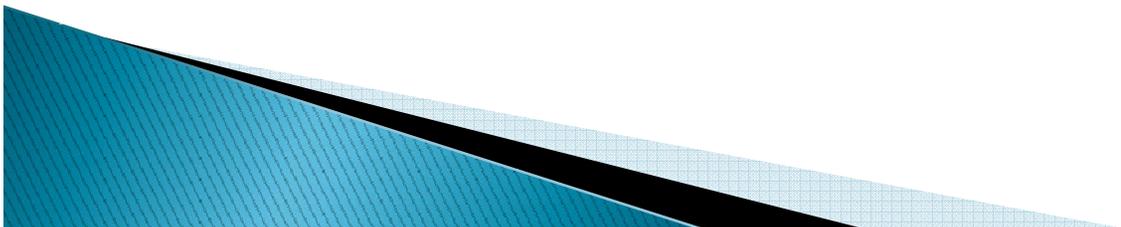
Duct Leakage Magnitude – RTUs

- **Light Commercial Buildings – RTU Systems**
 - Leakage Area data from Fan Pressurization
 - No measured pressures during normal operation
 - 364 systems – all located in Southern California
 - System size: average 3.9 tons (Std Deviation 2.1 tons)
 - Sample: Customers of two Commercial Service Contractors



Duct Leakage Magnitude – RTUs

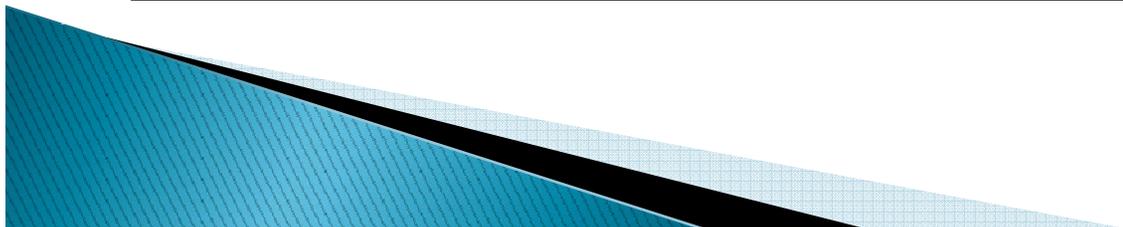
| Parameter | Leakage Area [cfm@25/ ton] | Fractional Leakage (Supply and Return) assuming 25 Pa avg at leaks | Fractional Leakage (Supply and Return) assuming 40 Pa avg at leaks |
|----------------------------------|---|---|---|
| Count [systems] | 364 | 364 | 364 |
| Average | 87 | 25% | 33% |
| Std Deviation [%] | 50% | 50% | 50% |
| Std Error in Mean [%] | 2.6% | 2.6% | 2.6% |



Energy Impact – RTUs

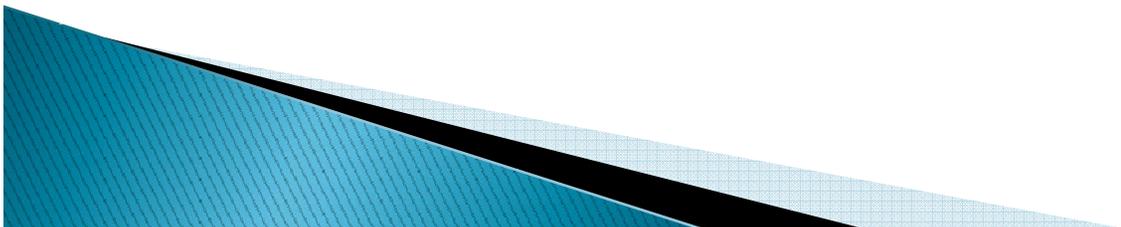
- **Approximate Analysis with ASHRAE Std 152**
 - Thermal losses are primary factor
 - Magnitude depends on location of ducts relative to insulation
 - For ducts above insulation, leakage split evenly between supply and return:

| | | |
|-------------------------|-----|-----|
| Duct Leakage | 25% | 33% |
| Heating Energy Increase | 16% | 23% |
| Cooling Energy Increase | 28% | 40% |
| Cooling Demand Increase | 44% | 68% |

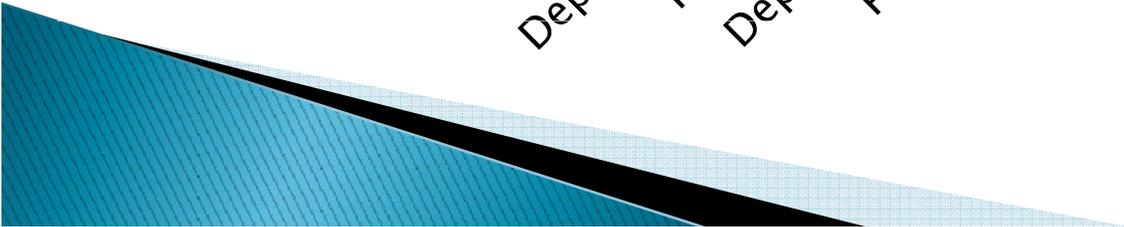
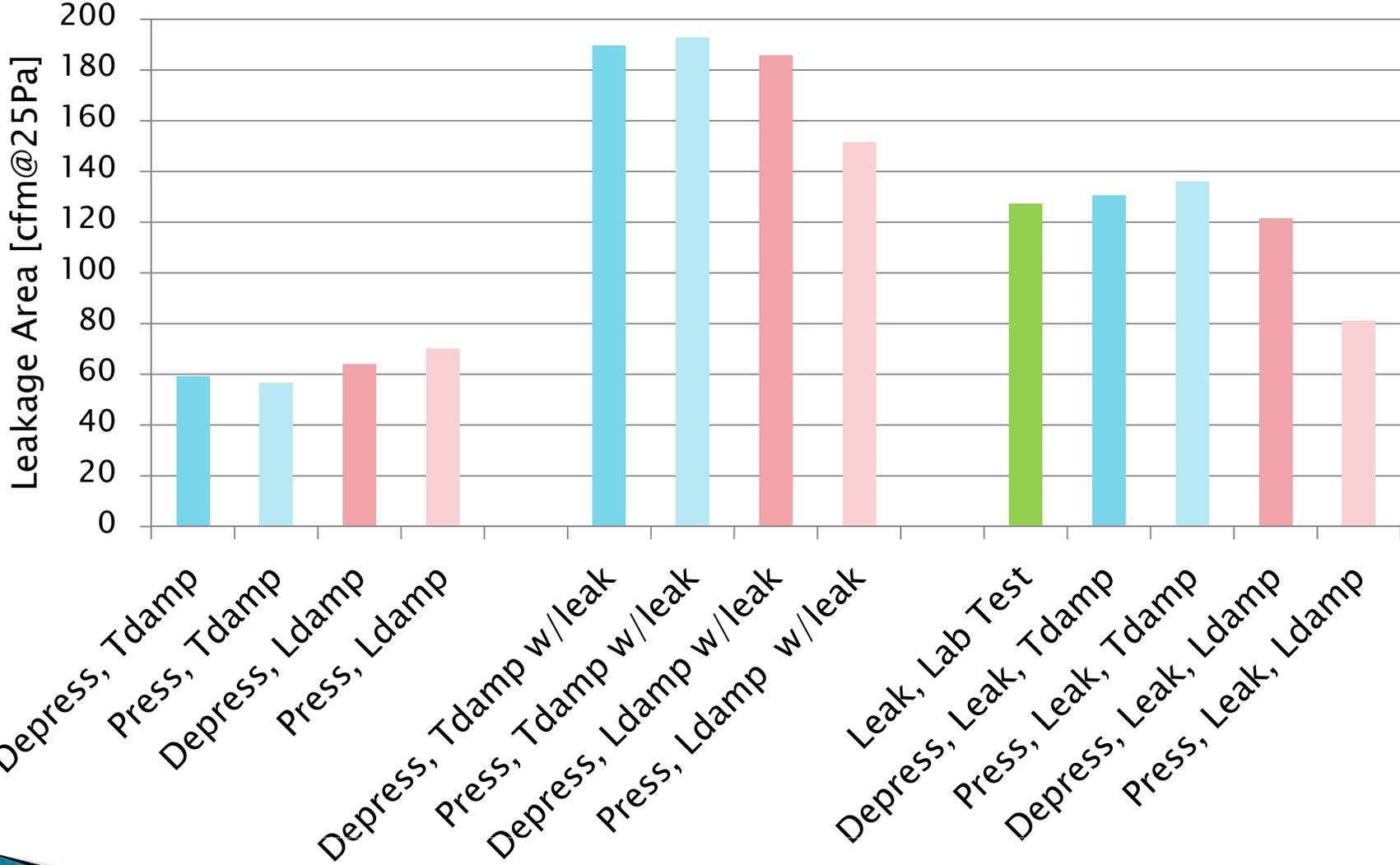


Leakage Diagnosis – VAV Supply

- **Leakage Downstream of VAV Boxes**
 - Standard fan pressurization
 - Test and Balance
 - Generally “low–pressure” ductwork and components
- **Simplified Diagnostic**
 - Leave fan running normally
 - Tape or block all grilles but one
 - VAV damper: close or set to minimum position
 - Pressurize/depressurize through open grille
 - Simultaneously calculate VAV damper opening and downstream leakage

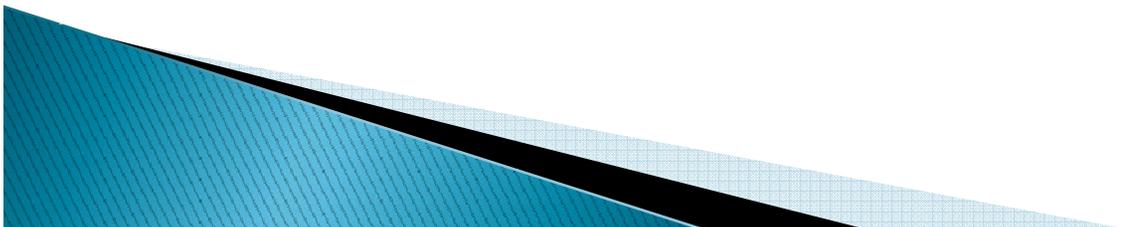


Simplified Diagnostic – VAV Supply



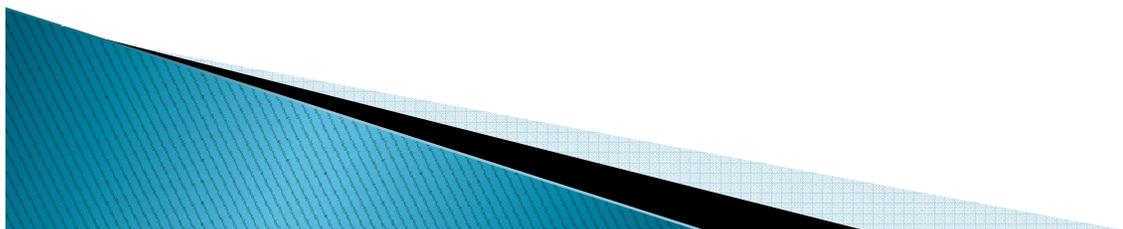
Leakage Magnitude – VAV Supply

- **Supply Leakage Downstream of VAV Boxes**
 - Leakage Area data from Simplified Diagnostic
 - Measured/estimated average downstream pressures during normal operation
 - 9 systems, located California, Florida, Rhode Island, Texas, Washington
 - Vintage: 1980s, 1990s
 - Sample: Office buildings, mostly military and university



Leakage Magnitude – VAV Supply

| Building | Flow Exponent | Best Estimate Leakage | Upstream ΔP [Pa] | Best Estimate ΔP leak [Pa] | Minimum (at grille) ΔP leak [Pa] | State |
|----------------------------|---------------|-----------------------|--------------------------|------------------------------------|--|-------|
| 1 | 0.80 | 8% | 250 | 25 | 8 | CA |
| 2 | 0.72 | 15% | 375 | 25 | 10 | WA |
| 3 | 0.54 | 14% | 300 | 25 | 20 | RI |
| 4 | 0.64 | 11% | 108 | 25 | 15 | RI |
| 5 | 0.61 | 19% | 550 | 50 | 50 | FL |
| 6 | 0.61 | 6% | 155 | 40 | 10 | TX |
| 7 | 0.78 | 4% | 155 | 40 | 10 | TX |
| 8 | 0.53 | 9% | 375 | 67 | 9 | CA |
| 9 | 0.41 | 6% | 488 | 50 | 20 | CA |
| Average | 0.63 | 10% | 306 | 39 | 17 | |
| Standard Dev [%] | 20% | 47% | 50% | 39% | 79% | |
| Std Err in Mean [%] | 7% | 16% | 17% | 13% | 26% | |



Energy Impact – VAV Supply

- **Fan Power**
 - Varies with flow rate raised to power 2.4 for typical supply systems
 - Duct leaks \Rightarrow short circuit of supply air to the return plenum \Rightarrow excess air flow through fan to meet loads
- **Heating/Cooling Energy**
 - Cooling load due to extra fan heat
 - Heating/cooling loads due to excess outdoor air
 - Ceiling loads during simultaneous heating and cooling
- **Implication of 10% supply leakage:**
 - **29%** excess fan power
 - **39%** including heating/cooling impact



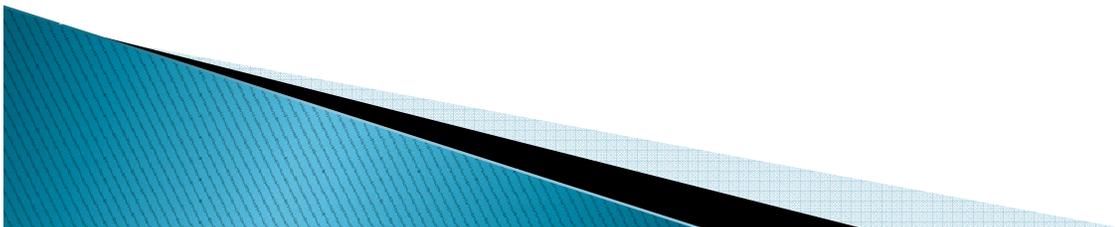
Leakage Diagnosis – Exhaust

- **Kitchen and Toilet Exhaust Systems**

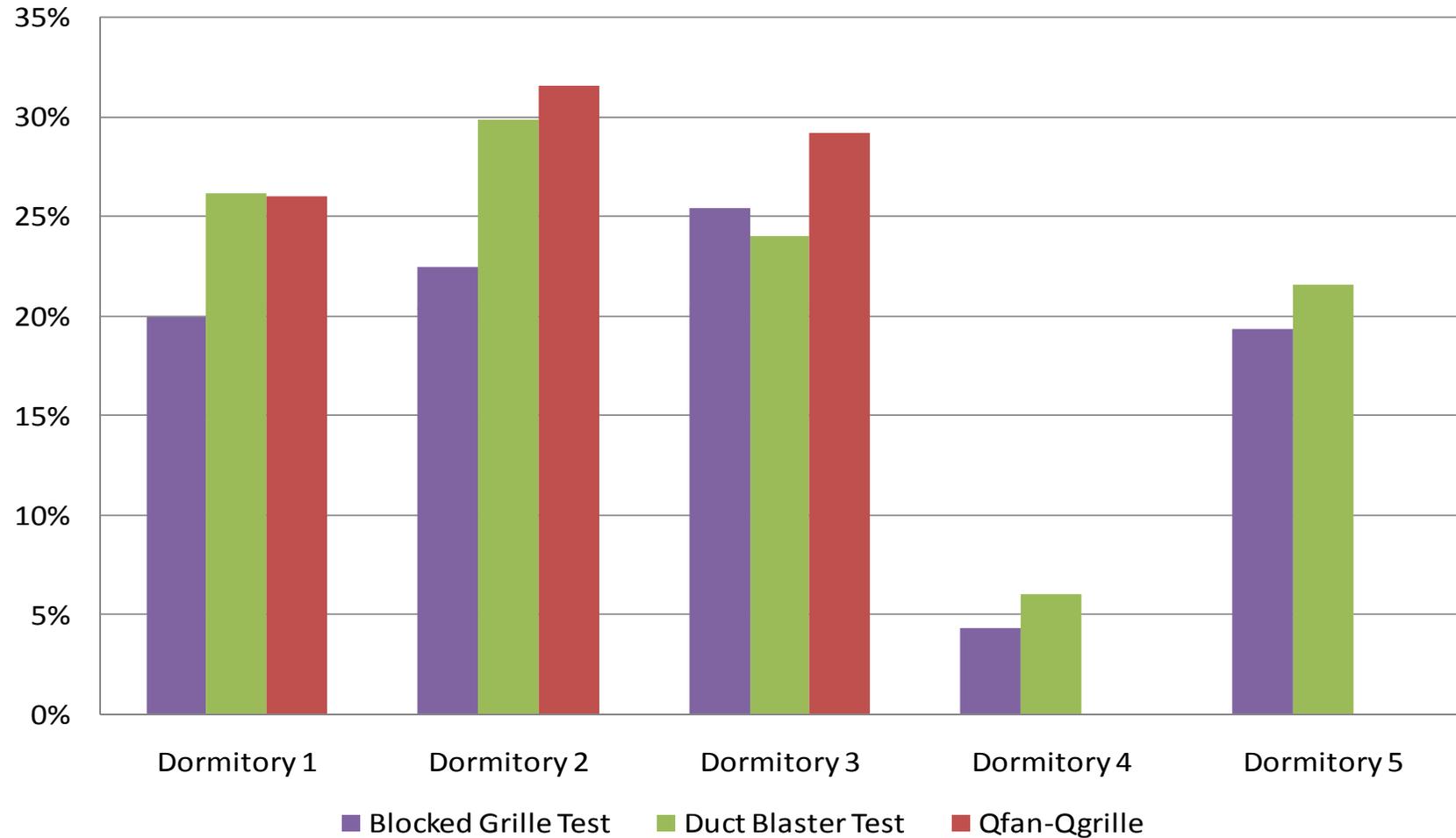
- Standard fan pressurization
- Test and Balance
- Generally “low–pressure”

- **Simplified Diagnostic**

- Leave fan running normally
- Tape or block all grilles
- Measure duct pressure at mid–point of shaft
- Measure flow leaving roof cap

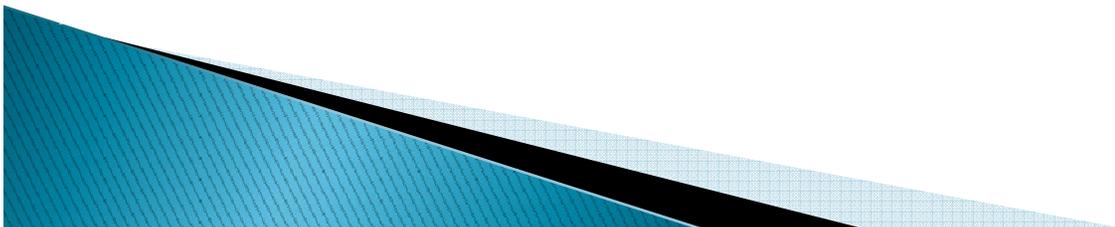


Bath Exhaust Leakage - Simplified Diagnostic



Kitchen/Bath Exhaust Leakage – “Standard” Fan Pressurization

| Building | Fan Flow [cfm] | Leakage [%] | Notes |
|---|----------------|-------------|--|
| Condominium (40-Story) | 950 | 74% | Building-Cavity Bathroom Exhaust |
| NYS University Dorm (10-story) | 2,300 | 70% | Bath/Shower Exhaust |
| NYS University Dorm (7-story) | 2,050 | 54% | Bath/Shower Exhaust |
| Navy BEQ (10-story dorm) | 6,470 | 54% | Building-Cavity Exhaust w/heat wheel |
| Barracks (eight 3-story buildings) | 20,000 | 20% | Bath/Shower Exhaust |
| Office Toilet Exhaust (3-story) | 8,700 | 9% | No pre-diagnosis of leakage |
| Seven NYC Apartment Exhausts | 2,450 | 36% | Kitchen/Bath Exhausts |
| Flow-Weighted AVERAGE for 20 Buildings | | 29% | Based upon leakage area and average pressure differential |



Energy Impacts: Kitchen/Bath Exhaust

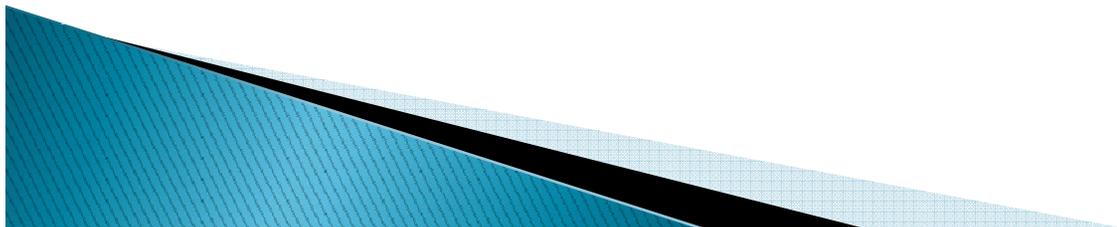
- **Fan Power**

- Scales with Cube of Flow
- **29%** Leakage Increases Fan Power by **179%** to Produce Design Flow at Grilles

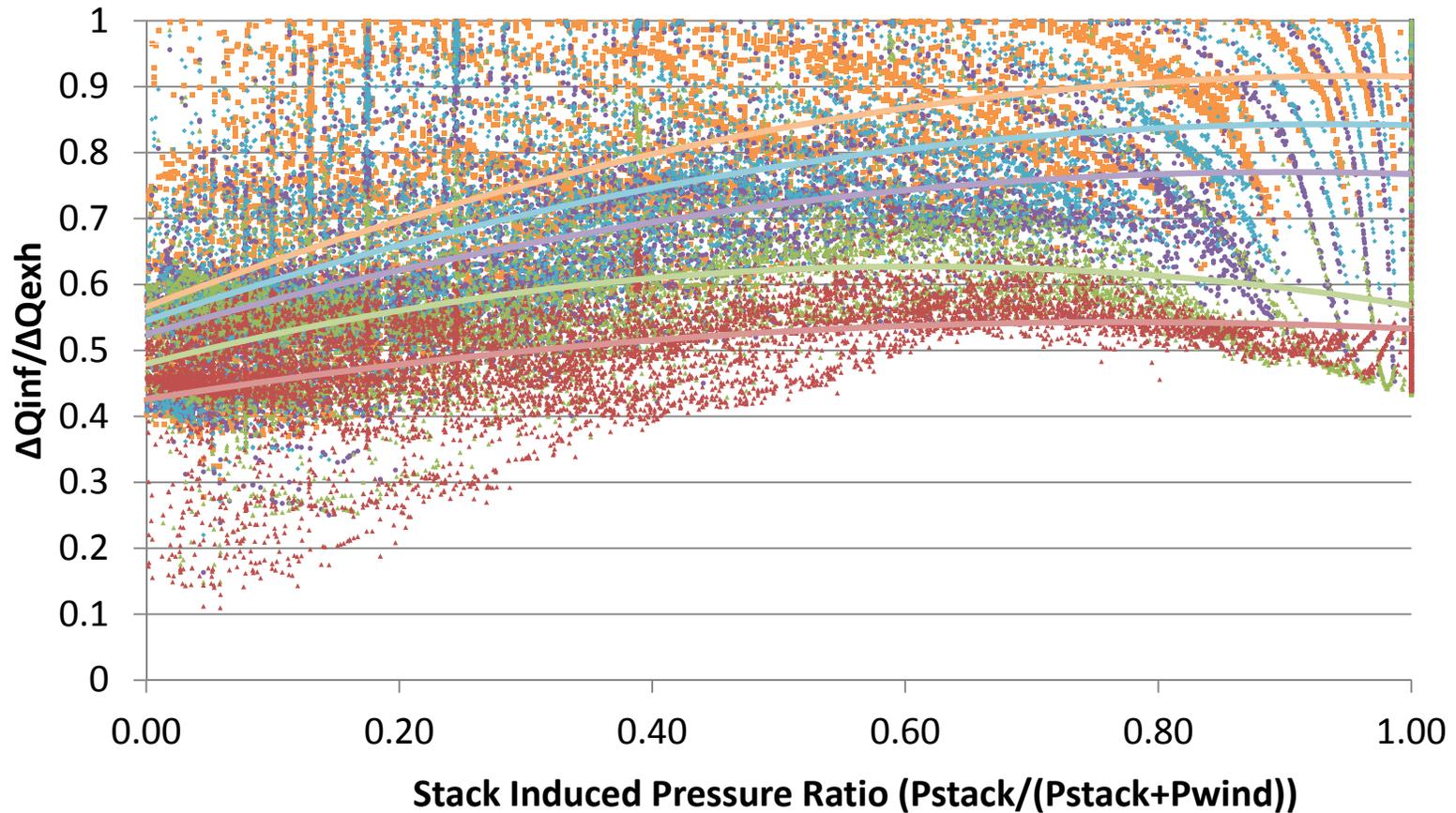
- **Heating and Cooling Loads**

- Scale with Excess Air Infiltration
- Change in infiltration is not equal to reduction in exhaust flow

$$\frac{\Delta Q_{\text{inf}}}{\Delta Q_{\text{exh}}} = \frac{Q_{\text{inf}}^{\text{initial}} - Q_{\text{inf}}^{\text{final}}}{Q_{\text{exh}}^{\text{initial}} - Q_{\text{exh}}^{\text{final}}}$$



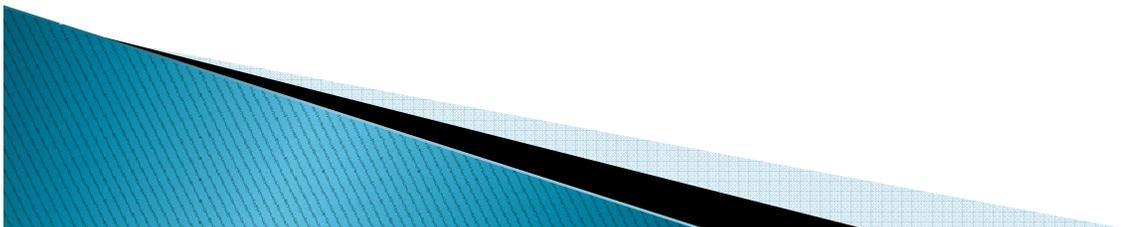
Exhaust Model Results: 6-Story Building



- $dQ_{exh}=1800 \text{ cfm}-1350 \text{ cfm}$
- $dQ_{exh}=1800 \text{ cfm}-900 \text{ cfm}$
- $dQ_{exh}=450 \text{ cfm}-225 \text{ cfm}$
- $n=2$ Polynomial Fit for 1350 cfm - 900 cfm
- Poly. ($dQ_{exh}=900 \text{ cfm}-450 \text{ cfm}$)
- $dQ_{exh}=1350 \text{ cfm} - 900 \text{ cfm}$
- $dQ_{exh}=900 \text{ cfm}-450 \text{ cfm}$
- $n=2$ Polynomial Fit for 1800 cfm - 900 cfm
- Poly. ($dQ_{exh}=450 \text{ cfm}-225 \text{ cfm}$)

Energy Impact – Kitchen/Bath Exhaust

- **Heating/Cooling Energy**
 - Assumptions
 - 75 cfm nominal kitchen plus bath exhaust for 1000 ft² apartment
 - 0.2 ACH natural infiltration
- **Implication of 29% exhaust leakage:**
 - **179%** excess fan power
 - **23%** increase in ventilation heating/cooling load



Energy Impact Summary

- **Light Commercial RTUs**

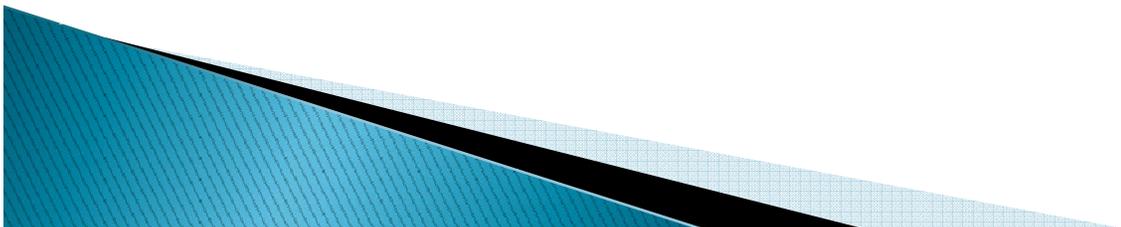
- 15%-70% increase in heating and cooling loads for ducts above ceiling insulation

- **Supply leakage downstream of VAVs**

- 30%-40% increase in system fan power

- **Kitchen and bath exhaust leakage**

- >150% increase in fan power
- >20% increase in infiltration load



Conclusions

- Low-pressure duct leakage is common and substantial
- Simplified techniques exist for measuring/diagnosing low-pressure leakage
- Energy impacts and analysis depend on type of system
 - Light Commercial RTUs
 - Supply leakage downstream of VAVs
 - Kitchen and bath exhaust leakage
- Importance of testing should be based upon leakage percentage of flow, not operating pressure

